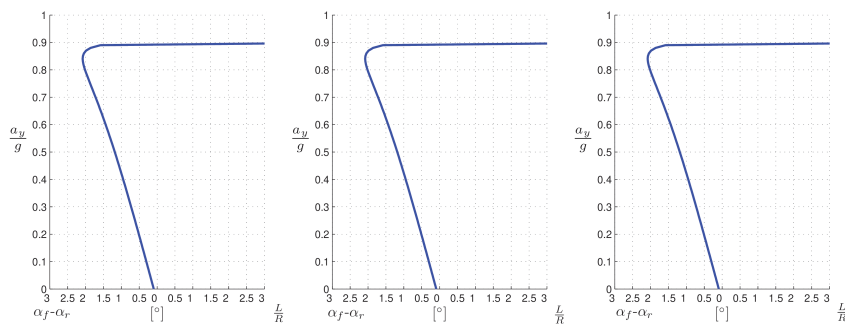
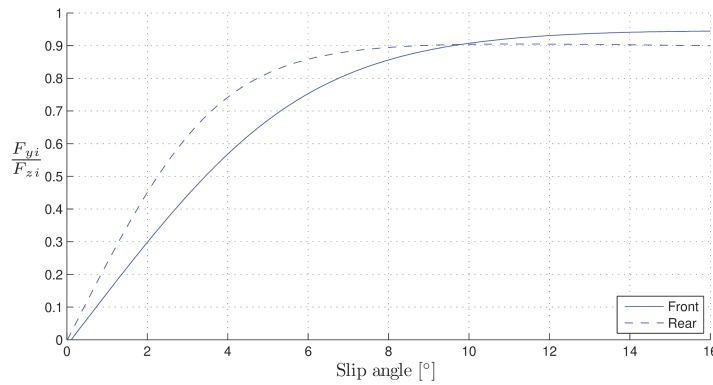


1. This is a continuation of exercise 5.1. The curves below show how  $\alpha_f - \alpha_r$  depends on  $a_y/g$ . Draw curves that allows you to determine the following:
  - a) The steer angle  $\delta_f$  for constant curve radius  $R = 100$  m.
  - b) The steer angle  $\delta_f$  for constant velocity 70 km/h.
  - c) The quotient  $L/R$  for constant steer angle  $\delta_f = 2.5^\circ$ .



2. This is a continuation of exercise 5.1. Assume that  $m = 1600$  kg,  $I_z = 2800$  kgm<sup>2</sup>,  $l_1 = l_2 = 1.4$  m, and that the center of gravity is low enough so that longitudinal load transfer may be neglected. The car is traveling at 70 km/h and keeps a constant curve radius of 100 m when a braking force is applied on the rear wheels.
  - a) Use the friction ellipse to determine  $F_{yr}$  if the braking force is  $F_x = 0.5 \cdot F_{x,max}$ .
  - b) Determine the yaw acceleration  $\dot{\Omega}_z$ .
  - c) How should  $\delta_f$  change for  $\dot{\Omega}_z = 0$  to hold instantaneously?

The figure below is the tire-force characteristics from exercise 5.1.



3. In Lecture 4 the equation

$$\delta_f = \frac{L}{R} + K_{us} \frac{a_y}{g}$$

was derived for the steering angle at steady state cornering in the case of front-wheel steering. Assume that a rear-wheel steering is added to the model with the steer angle  $\delta_r$ , as we did in Lecture 7. What is the corresponding relation?

4. It was shown in Lecture 7 that

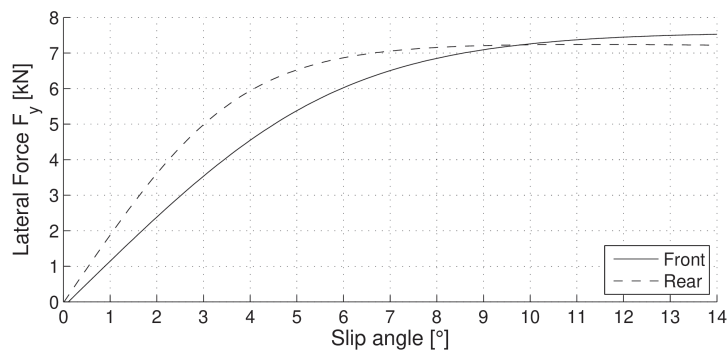
$$\begin{aligned} \begin{bmatrix} V_y(s) \\ \Omega_z(s) \end{bmatrix} &= (sM + A)^{-1}(\mathbf{u}_f \delta_f(s) + \mathbf{u}_r \delta_r(s)) \\ &= \frac{1}{\Delta} \begin{bmatrix} I_z s + a_4 & -a_2 \\ -a_3 & m s + a_1 \end{bmatrix} \left( \begin{bmatrix} 2C_{\alpha f} \\ 2l_1 C_{\alpha f} \end{bmatrix} \delta_f + \begin{bmatrix} 2C_{\alpha r} \\ -2l_2 C_{\alpha r} \end{bmatrix} \delta_r \right) \end{aligned}$$

where

$$\Delta = I_z m s^2 + (I_z a_1 + m a_4) s + (a_1 a_4 - a_2 a_3).$$

Assume that  $C_{\alpha f} = C_{\alpha r} = C_\alpha$  and  $l_1 = l_2 = L/2$  and consider a step in the front steer angle,  $\delta_f(s) = 1/s$ . Use the initial value theorem, i.e.,  $f(0^+) = \lim_{s \rightarrow \infty} sF(s)$ , to determine the immediate response in  $\dot{V}_y$  and  $\Omega_z$  in the two cases  $\delta_r = \delta_f$  (rear wheels in phase) and  $\delta_r = -\delta_f$  (rear wheels out-of phase), respectively. (Hint: You don't need to calculate the matrix  $A$ .)

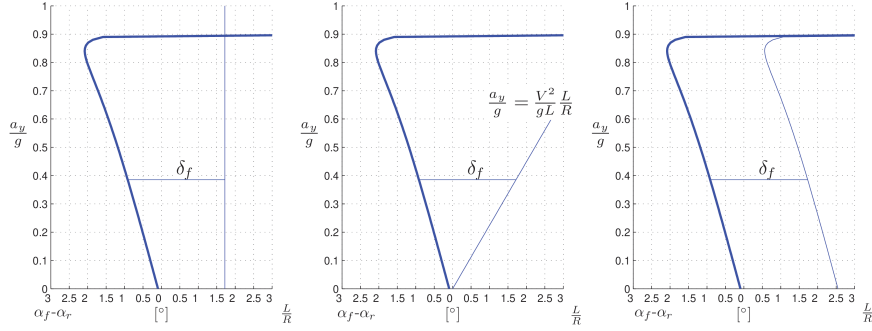
5. The lateral forces, as a function of the slip angle, for the front and rear tires respectively is given by the following figure:



The vehicle mass is  $m = 1600$  kg, the wheelbase  $L = 2.6$  m, and the center of gravity is 1.2 m behind the front axle. The vehicle is driving through a long curve with radius is  $R = 100$  m (assume stationary conditions). Determine the steer angle  $\delta_f$  if the velocity is  $v = 90$  km/h.

## Answers

### 1. Curves:



2. a)  $F_{yr} = 2.6 \text{ kN}$   
 b)  $\dot{\Omega}_z = 0.20 \text{ rad/s}^2$   
 c)  $\delta_f$  should be reduced by about  $0.2^\circ$ .

3.  $\delta_f - \delta_r = \frac{L}{R} + K_{us} \frac{a_y}{g}$

4. if  $\delta_f = \delta_r$ :

$$\begin{bmatrix} V_y(0^+) \\ \Omega_z(0^+) \end{bmatrix} = \begin{bmatrix} \frac{4C_\alpha}{m} \\ 0 \end{bmatrix}$$

if  $\delta_f = -\delta_r$ :

$$\begin{bmatrix} V_y(0^+) \\ \Omega_z(0^+) \end{bmatrix} = \begin{bmatrix} 0 \\ \frac{2LC_\alpha}{I_z} \end{bmatrix}$$

5.  $\alpha_f \approx 5^\circ$ ,  $\alpha_r \approx 2.6^\circ$ ,  $\delta_f = 3.9^\circ$